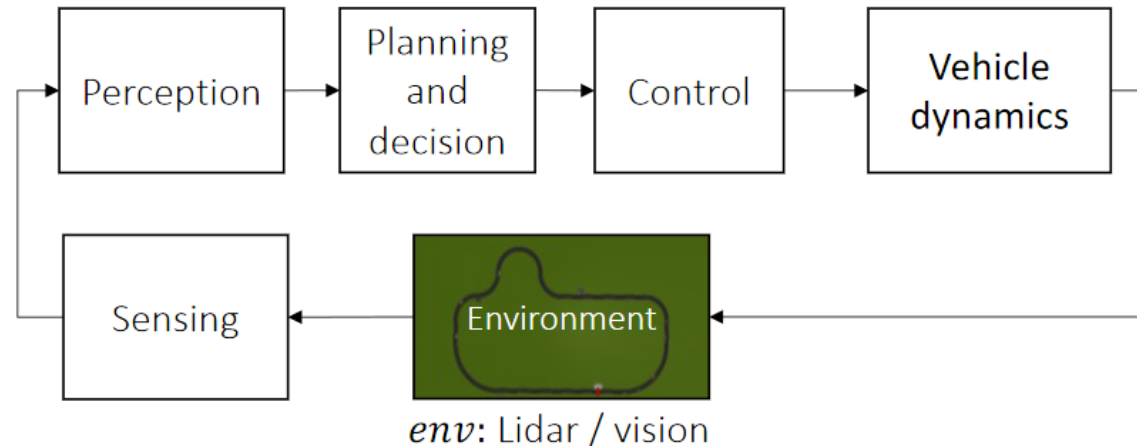


Control

# Control Problem

The Control module sits between higher-level decisions (e.g., changing lanes, slowing down) and the lower-level control and actuation (e.g., steering and throttle).

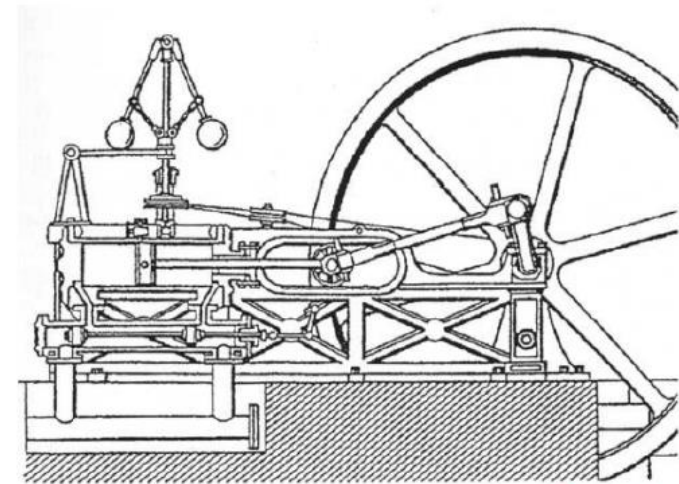
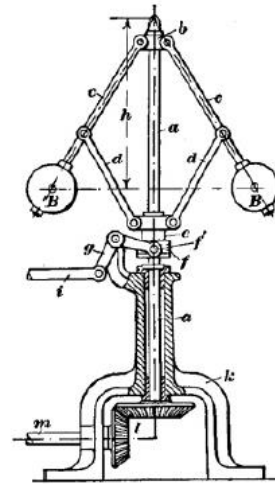


# Control Theory

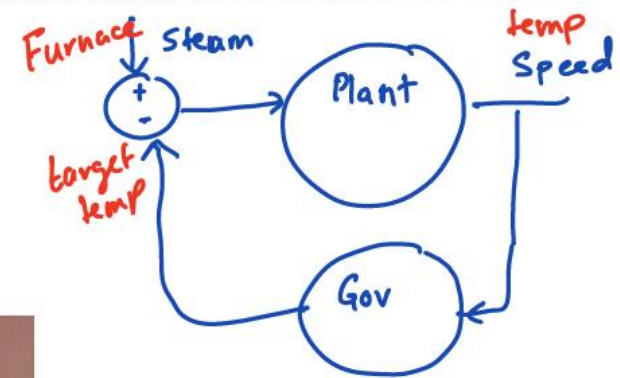
- Control theory is the art of making things do what you want them to do.
- Art describes the creation of parameterized controllers or algorithms and ways of tuning them.
- By things we mean phenomena that can be represented using differential equations.
  - we want them to follow some desired behavior, track some set-point, or follow a reference trajectory.  
Requirement, invariants

# Example:

Feedback



Feed forward  
-ve Feed back



(a) F/A-18 "Hornet"



(b) X-45 UCAV

# Examples of Control System:

- Thermostat-heating system: the temperature is kept within a desirable range, despite the changes in occupancy and weather conditions.
- Cruise control: The car maintains a set reference speed, despite changes in slope, road conditions.
- Autosteer: The car steers to track the center of the lane markings.

# Roadmap

## **Modeling the control problem using Differential Equations**

- Solutions and their properties

## **Control design**

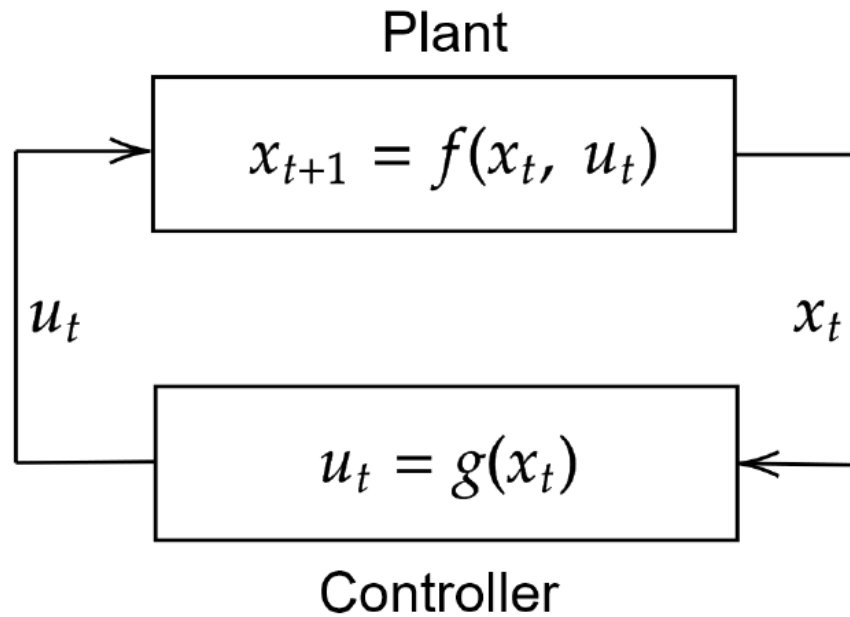
- Open vs. closed loop control
- PID control design
- State feedback

## **Requirements**

- Stability
- Lyapunov theory and its relation to invariance

# Differential Equation Model:

A control system can be modeled using two components:  
a plant and a controller.



# Continuous Version

The continuous time version of this model is written using *ordinary differential equations (ODE)*:

$$\dot{x}(t) = f(x(t), u(t)), \text{ or}$$

$$\dot{x}(t) = f'(x(t))$$

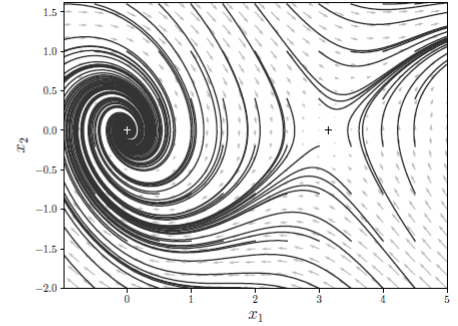
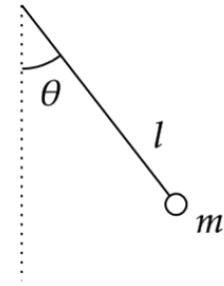
A Solution to ODE is:



# Examples of ODE

ODE describing the pendulum system is the following:

$$\begin{bmatrix} \dot{x}_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ \frac{g}{l} \sin(x_1) - \frac{k}{m} x_2 \end{bmatrix}$$



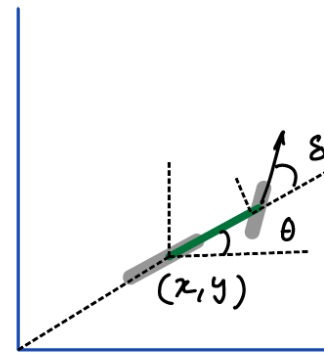
How do we find equilibria:

# Vehicle Model

## Example 2 Bicycle / Kinematic Vehicle model

Ref. Brian Paden et al. 2016

Survey of motion planning and control for self-driving



$v$ : longitudinal velocity

$x, y$ : position of rear wheel on the plane

$\theta$ : heading angle

$[x, y, \theta, v]$ : state

$\delta$ : steering angle  
control input

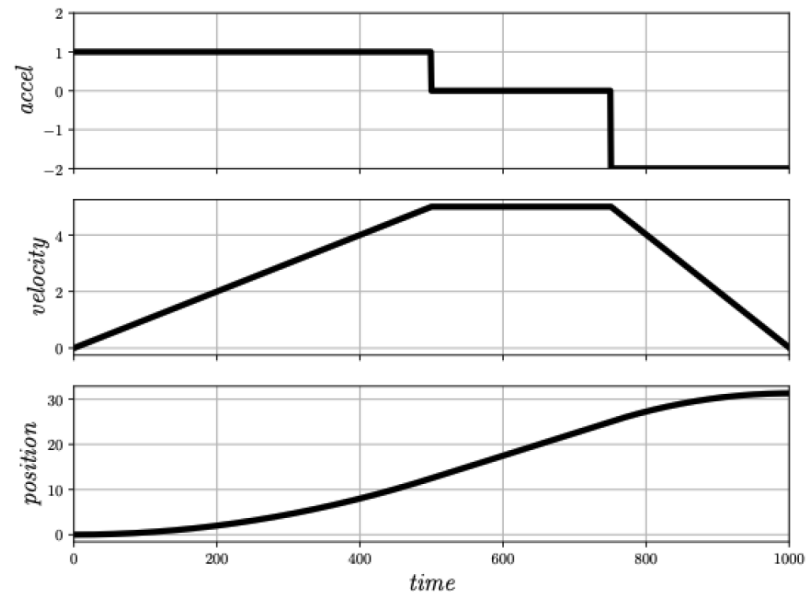
$l$ : length of bicycle parameter

Equations of motion derived in the above paper

$$\left. \begin{aligned} \dot{x}(t) &= v(t) \cos \theta(t) \\ \dot{y}(t) &= v(t) \sin \theta(t) \\ \dot{\theta}(t) &= \frac{v(t)}{l} \tan \delta(t) \end{aligned} \right\} \dot{x} = f(x, u)$$

# When do the Solutions Exist?

- If the control input  $u(t)$  is not continuous, then the solution  $x(t)$  will not be differentiable (at the points of discontinuity). See Figure below we have to be careful about the definition of solution.



# When do the Solutions Exist?

Example  $\dot{x} = x^2$  with  $x(0) = 1$   $x(t)$

Check that  $x(t) = \frac{1}{1-t}$  is a solution

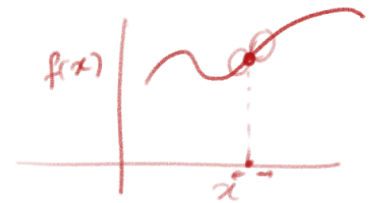
$$\frac{dx(t)}{dt} = -\frac{1}{(1-t)^2} \cdot (-1) = \frac{1}{(1-t)^2} = x^2(t) \quad \checkmark$$

But as  $t \rightarrow 1$   $x(t) \rightarrow \infty$  blows up

Additional assumption on  $f$

$f$  should not grow too fast

intuition

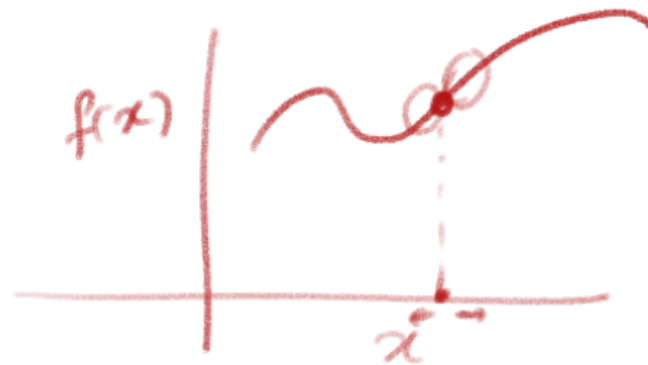


# When do the Solutions Exist?

Additional assumption on  $f$

$f$  should not grow too fast

intuition



# Lipschitz Continuous

**Definition 5.1.** A function  $f : \mathbb{R}^n \rightarrow \mathbb{R}^n$  is *Lipschitz continuous* if  $\exists L > 0$  such that for any pair  $x, x' \in \mathbb{R}^n$ ,

$$\|f(x) - f(x')\| \leq L\|x - x'\|.$$

# Lipschitz Continuous

Example .  $f(x) = 6x + 4$

All differentiable functions with bounded derivative are Lipschitz Continuous

$|x|$  not differentiable but Lipschitz

$x^2$  not Lipschitz  $\|f(x) - f(x')\|$   
 $= \|x^2 - x'^2\|$

Suppose  $\exists L$  such that  $\|x_1^2 - x_2^2\| \leq L(x_1 - x_2)$

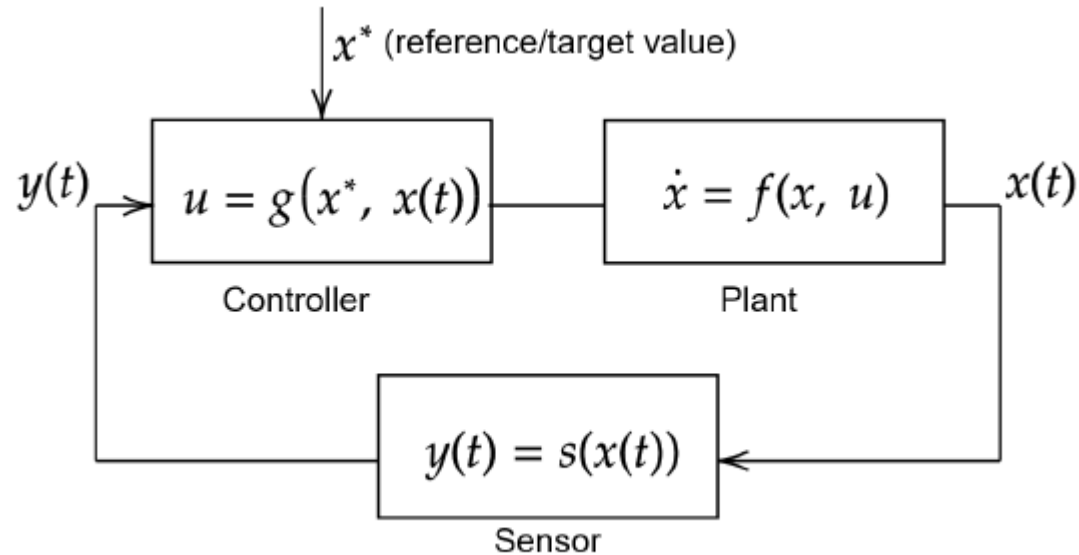
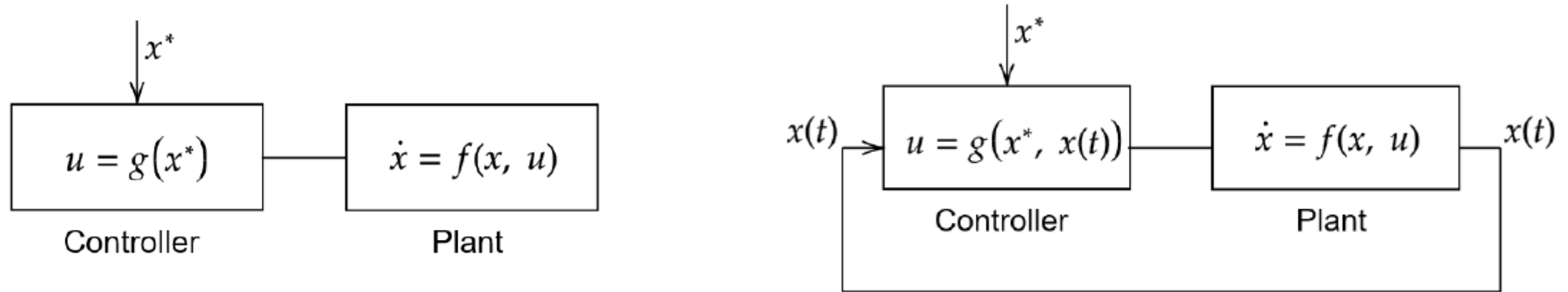
# Lipschitz Continuous

$\sqrt{x}$  is also not Lipschitz  $\dot{x} = f(x) \mid \dot{x} = f(x, u)$

Thm. If  $f(x, u)$  is Lipschitz continuous in the first argument and  $u(t)$  is piecewise continuous then  $\dot{x} = f(x, u)$  has unique solutions.

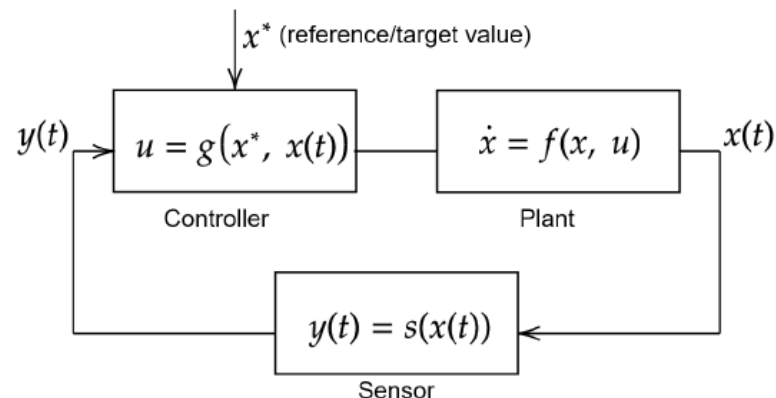


# Open Loop and Closed Loop Control



# Proportional Control Example

One example is where a car driving down a lane on the road. If the car is very close to the left edge of the lane, we would want a considerable turn in the right direction. If the car is slightly to the left, we would want to move just a little bit to the right. Here the error is the deviation of the car's position  $x(t)$  from the center of the lane  $x^*$ . This motivates the incorporation of proportional control, where we can include a proportional gain term as so:  $u(t) = g(x^*, x(t)) = K_p(x^* - x(t))$ . This is also known as error feedback control, and to show this in our model, we add a sensor, as seen in the following Figure



# PID Control

There are more ways to fine-tune the function  $g$ :

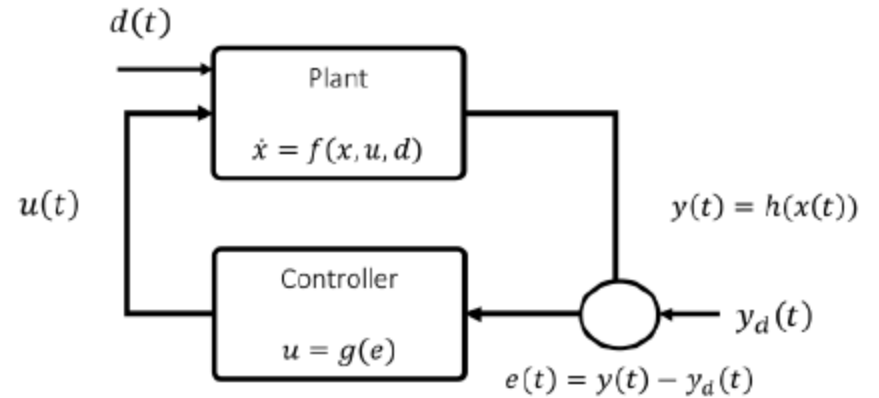
- $g$  not only depends on the error but also the rate of change of error (derivative)
- $g$  also depends on the history of the error (integral)

This gives the general form of the PID controller:

$$\begin{aligned}u(t) &= K_P e(t) + K_I \int_0^t e(\tau) d\tau + K_D \frac{de(t)}{dt} \\ &= K_P \left[ e(t) + \frac{1}{T_I} \int_0^t e(\tau) d\tau + T_D \frac{de(t)}{dt} \right]\end{aligned}$$

# A simple P-controller example

- $\dot{y}(t) = u(t) + d(t)$
- Using proportional (P) controller
- $u(t) = -K_P e(t) = -K_P (y(t) - y_d(t))$
- $\dot{y}(t) = -K_P y(t) + K_P y_d(t) + d(t)$
- Consider constant setpoint  $y_0$  and disturbance  $d_{ss}$
- $\dot{y}(t) = -K_P y(t) + K_P y_0 + d_{ss}$
- What is the steady state output?
  - Set  $-K_P y(t) + K_P y_0 + d_{ss} = 0$
  - $y(t) = y_{ss} = \frac{d_{ss}}{K_P} + y_0$



# A simple P-controller example

- $\dot{y}(t) = u(t) + d(t)$
- Consider constant setpoint  $y_0$  and disturbance  $d_{ss}$
- $\dot{y}(t) = -K_P y(t) + K_P y_0 + d_{ss}$
- Steady state output  $y(t) = y_{ss} = \frac{d_{ss}}{K_P} + y_0$
- Transient behavior
  - $y(t) = y(0)e^{-t/T} + y_{ss} \left(1 - e^{-\frac{t}{T}}\right)$ ,  $T = 1/K_P$
- To make steady state error small we can increase  $K_P$  at

