### ECE484 Principles of Safe Autonomy Lecture 7 Modeling and Control Sayan Mitra

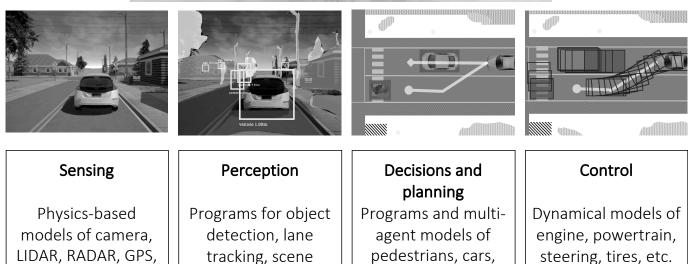


GEM platform

### Autonomy pipeline

etc.

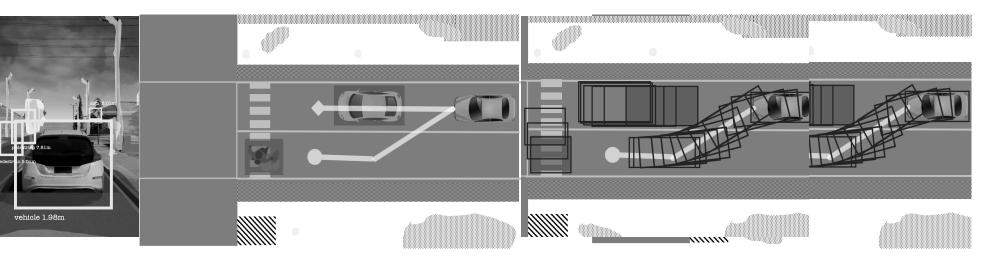




etc.

understanding, etc.





**Control** Dynamical models of engine, powertrain, steering, tires, etc.



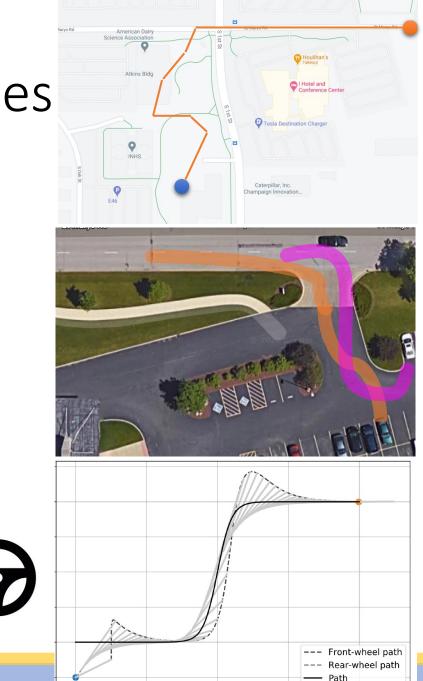
### Outline

- Modeling the control problem
  - Differential Equations; solutions and their properties
- Control design
  - Open loop vs closed loop
  - PID
  - State feedback
  - MPC (brief)
- Requirements
  - Stability
  - Lyapunov theory and its relation to invariance



### Typical planning and control modules

- Global navigation and planner
  - Find paths from source to destination with static obstacles
  - Algorithms: Graph search, Dijkstra, Sampling-based planning
  - Time scale: Minutes
  - Output: reference center line, does not consider vehicle dynamics
- Local planner
  - Dynamically feasible trajectory generation
  - Dynamic planning w.r.t. obstacles
  - Time scales: 10 Hz
- Controller
  - Waypoint follower using steering, throttle
  - Algorithms: PID control, MPC, Lyapunov-based controller
  - Lateral/longitudinal control
  - Time scale: 100 Hz





### What is control

Control theory is the *art* of making *things* do what *you want* them to do

art: tuning parameters things: Differential equation models what you want: tracking error or stability

### Open look control



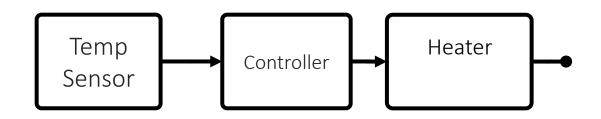
System: Sensor, control logic, heater

Control logic: Check every 30 mins

If temperature  $\theta_s \leq 70$  then run heater for the next 30 mins;

if  $\theta_s \ge 75$  then turn off heater for the next 30 mins

**Open loop**: output of the system is not used by the controller



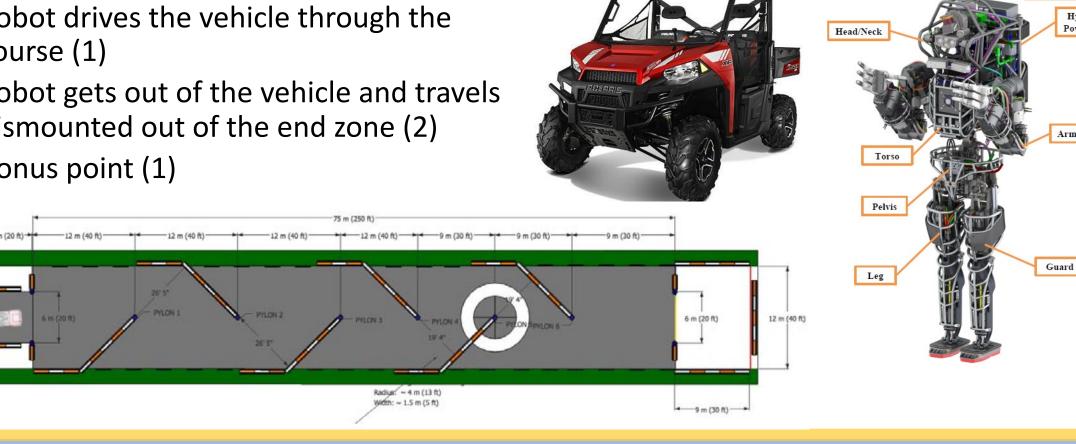


### Complex control tasks: DARPA Robotics Challenge

4 points task

12 m (40 ft)

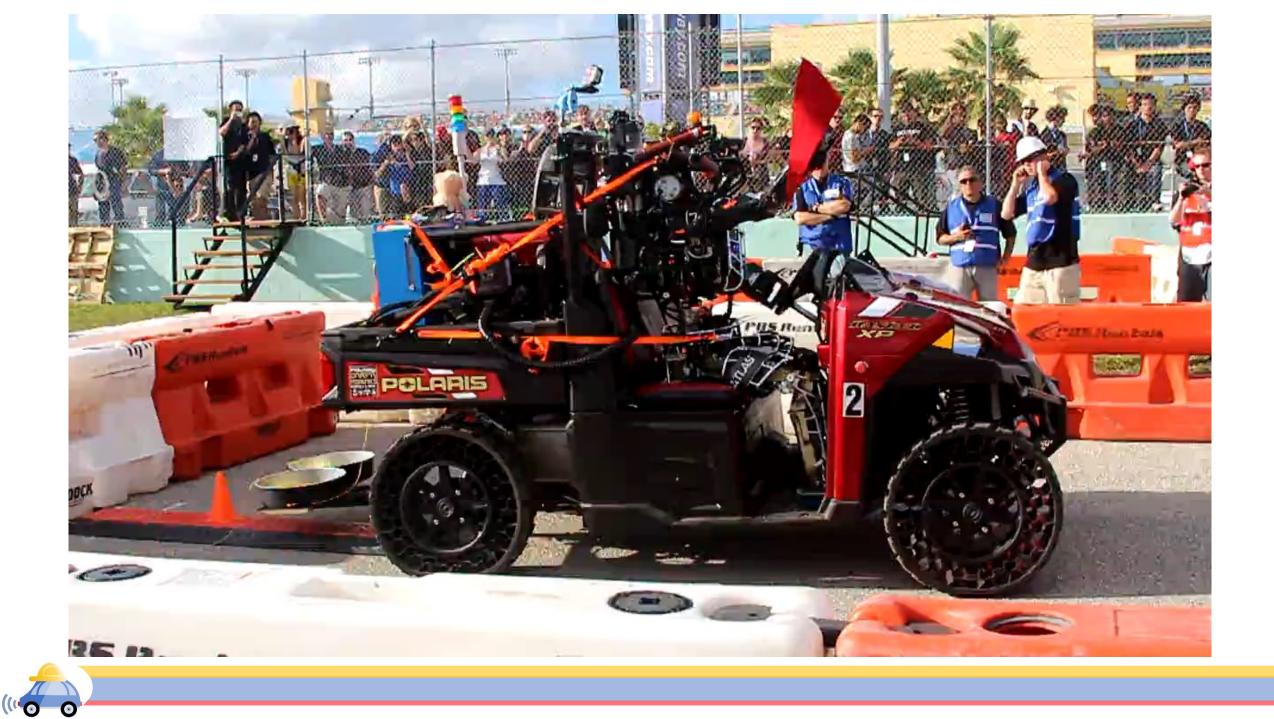
- Robot drives the vehicle through the course (1)
- Robot gets out of the vehicle and travels dismounted out of the end zone (2)
- Bonus point (1)

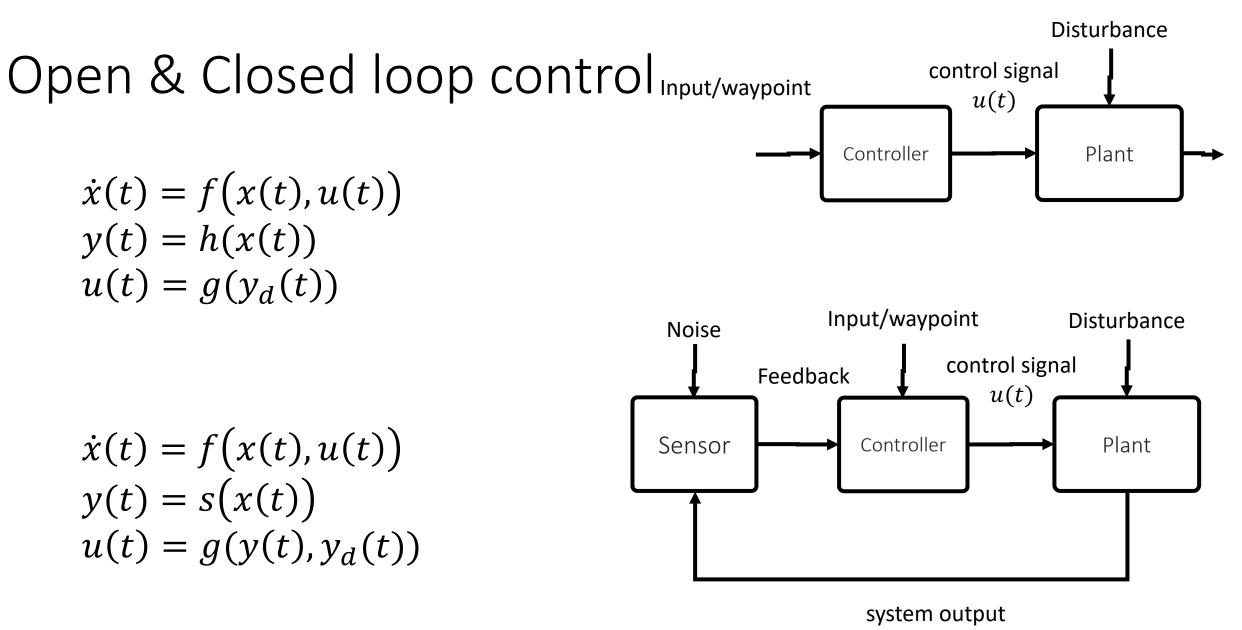


Hoist Mount Points

Arm

Hydraulic Power Pack





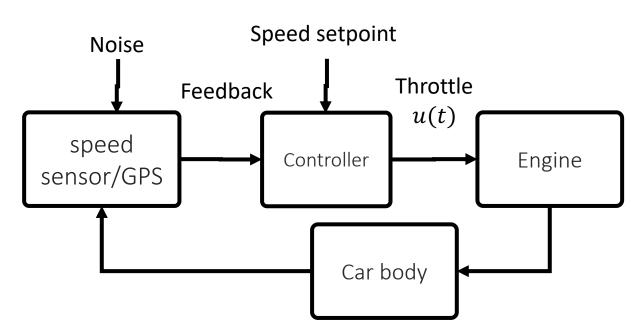
### Cruise control

$$\dot{x}(t) = f(x(t), u(t))$$
  

$$y(t) = s(x(t))$$
  

$$u(t) = g(y(t), y_d(t))$$

Control design is the problem of figuring out g given certain requirements on y(t)







### Modeling control systems

Behaviors of physical processes are described in terms of instantaneous laws

Common notation: 
$$\frac{dx(t)}{dt} = f(x(t), u(t), t) - (1),$$
  
where time  $t \in \mathbb{R}$ ; state  $x(t) \in \mathbb{R}^n$ ; input  $u(t) \in \mathbb{R}^m$ ;  $f: \mathbb{R}^n \times \mathbb{R}^m \times \mathbb{R} \to \mathbb{R}^n$ 

Example. 
$$\frac{dx(t)}{dt} = v(t)$$
;  $\frac{dv(t)}{dt} = -g$ 

Initial value problem: Given system (1) and initial state  $x_0 \in \mathbb{R}^n$ ,  $t_0 \in \mathbb{R}$ , and input u:  $\mathbb{R} \to \mathbb{R}^m$ , find a state trajectory or *solution* of (1).



Lecture Slides by Sayan Mitra mitras@illinois.edu

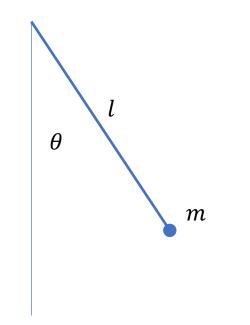
### Example 1: Pendulum

Pendulum equation

$$x_1 = \theta \ x_2 = \dot{\theta}$$
$$x_2 = \dot{x}_1$$
$$\dot{x}_2 = -\frac{g}{l}\sin(x_1) - \frac{k}{m}x_2$$

k: friction coefficient

 $\begin{bmatrix} \dot{x_2} \\ \dot{x_1} \end{bmatrix} = \begin{bmatrix} -\frac{g}{l}\sin(x_1) - \frac{k}{m}x_2 \\ x_2 \end{bmatrix}$ 

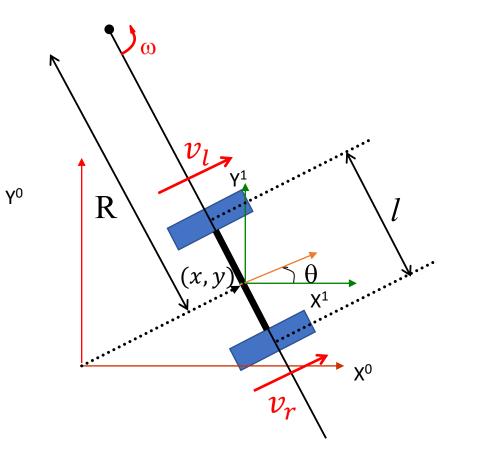


#### What is described?

-> Center of mass movement relative to the origin



### Example 2. Differential Drive Model



Instantaneous Center of Curvature =  $[x - R \sin \theta, y + R \cos \theta] = [ICC_x, ICC_y]$ 

$$\omega(R + l/2) = v_r$$
  

$$\omega(R - l/2) = v_l$$
  

$$R = \frac{l}{2} \frac{(v_r + v_l)}{(v_r - v_l)}$$
  

$$\omega = \frac{v_r - v_l}{l}$$



### Example 3: Simple vehicle model: Dubin's car

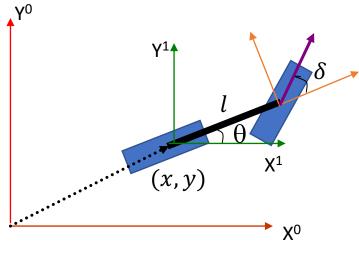
#### Key assumptions

- Front and rear wheel in the plane in a stationary coordinate system
- Steering input, front wheel steering angle  $\delta$
- No slip: wheels move only in the direction of the plane they reside in

Zeroing out the accelerations perpendicular to the plane in which the wheels reside, we get the equations in the next slide

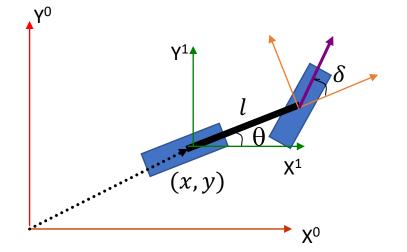
#### Modeling one wheel is enough

Reference: Paden, Brian, Michal Cap, Sze Zheng Yong, Dmitry S. Yershov, and Emilio Frazzoli. 2016. A survey of motion planning and control techniques for self-driving urban vehicles. IEEE Transactions on Intelligent Vehicles 1 (1): 33–55.





### Rear Wheel Model (Dubin's model)



Car length = lCar (real wheel) pose =  $\begin{bmatrix} x \\ y \\ \theta \end{bmatrix}$ Car speed = vCar (front wheel) steering angle =  $\delta$ 

$$\dot{x} = v \cos\theta$$
$$\dot{y} = v \sin\theta$$
$$\dot{\theta} = \frac{v}{l} \tan\delta$$



### Notions of solution

What is a solution? Many different notions.

**Definition 1.** (First attempt) Given  $x_0$  and u,  $\xi \colon \mathbb{R} \to \mathbb{R}^n$  is a solution of trajectory iff

(1) 
$$\xi(t_0) = x_0$$
 and  
(2)  $\frac{d}{dt}\xi(t) = f(\xi(t), u(t), t)), \forall t \in \mathbb{R}.$ 

Mathematically OK, but too restrictive for autonomous systems.

Assumes that  $\xi$  is not only continuous, but also differentiable. This disallows u(t) to be discontinuous, which is often required for optimal control.



time

# Is PC input u(t) adequate for guaranteeing existence of solutions?

Example.  $\dot{x}(t) = -sgn(x(t)); x_0 = c; t_0 = 0; c > 0$ Solution:  $\xi(t) = c - t$  for  $t \le c$ ; check  $\dot{\xi}(t) = -1 = -sgn(\xi(t))$ Problem: Solution undefined at t = c, f discontinuous in x

Example. 
$$\dot{x}(t) = x^2$$
;  $x_0 = c$ ;  $t_0 = 0$ ;  $c > 0$   
Solution:  $\xi(t) = \frac{c}{1-tc}$  works for  $t < 1/c$ ; check  $\dot{\xi}$   
Problem: As  $t \to \frac{1}{c}$  then  $x(t) \to \infty$ ;  $p$  grows too fast

No, we need stronger conditions on smoothness of f(.)



### Lipschitz continuity

A function  $f: \mathbb{R}^n \to \mathbb{R}$  is Lipschitz continuous if there exist L > 0 such that for any pair  $x, x' \in \mathbb{R}^n$ ,  $||f(x) - f(x')|| \le L||x - x'||$ 

Examples: 6x + 4; |x|; all differentiable functions with bounded derivatives

Are Lipschitz continuous functions closed under addition, multiplication?

Non-examples:  $\sqrt{x}$ ;  $x^2$  (locally Lipschitz)



### Dynamical Systems Model

Describe behavior in terms of instantaneous laws  $\frac{dx(t)}{dt} = f(x(t), u(t))$ 

 $t \in \mathbb{R}, x(t) \in \mathbb{R}^n, u(t) \in \mathbb{R}^m$ 

 $f: \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^n$ dynamic function

**Theorem.** If f(x(t), u(t)) is Lipschitz continuous in the first argument and u(t) is piece-wise continuous then (1) has unique solutions.



### Modified notion of solution\*

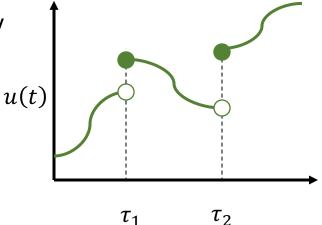
Definition.  $u(\cdot)$  is a *piece-wise continuous* with set of discontinuity points  $D \subseteq \mathbb{R}^m$  if

- (1)  $\forall \tau \in D, \lim_{t \to \tau^+} u(t) < \infty; \lim_{t \to \tau^-} u(t) < \infty$
- (2) Continuous from right  $\lim_{t \to \tau^+} u(t) = u(t)$
- (3)  $\forall t_0 < t_1$ ,  $[t_0, t_1] \cap D$  is finite

 $PC([t_0, t_1], \mathbb{R}^m)$  is the set of all piece-wise continuous functions over the domain  $[t_0, t_1]$ 

**Definition 2**. Given  $x_0$  and u,  $\xi \colon \mathbb{R} \to \mathbb{R}^n$  is a solution or trajectory iff (1)  $\xi(t_0) = x_0$  and (2)  $\frac{d}{dt}\xi(t) = f(\xi(t), u(t), t), \forall t \in \mathbb{R} \setminus \mathbb{D}$ .

Since u(t) is piece-wise continuous, so is f in the second argument



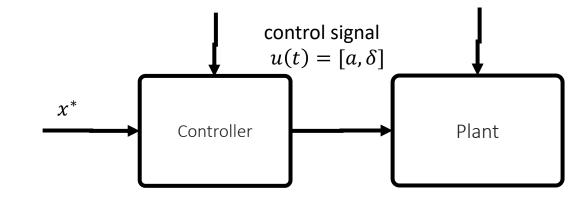


## Control design



### Open loop control

Disturbance





### PID control strategy

- Error measured as difference between desired output and measured output
  - $e(t) = y_d(t) y(t)$
- Control input defined as terms proportional to error, integral of error, and derivative of error

• 
$$u(t) = K_P e(t) + K_I \int_0^t e(\tau) d\tau + K_D \frac{de(t)}{dt}$$
  
=  $K_P \left( e(t) + \frac{1}{T_I} \int_0^t e(\tau) d\tau + T_D \frac{de(t)}{dt} \right)$ 

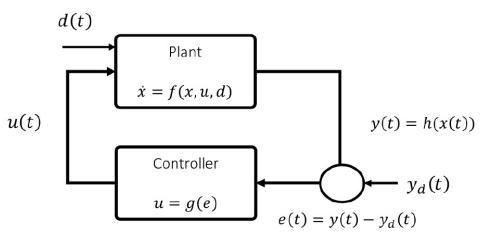
- It is necessary to tune these gains of the PID controller
  - If the gain is too high the system may become unstable
  - If the gain is too low the system may not respond
- PD control:  $K_I = 0$  or  $T_I = \infty$
- PI control:  $K_D = 0 = T_D$
- P control:  $K_I = 0$  and  $K_D = 0$ 
  - Steady state may not be 0



### A simple P-controller example

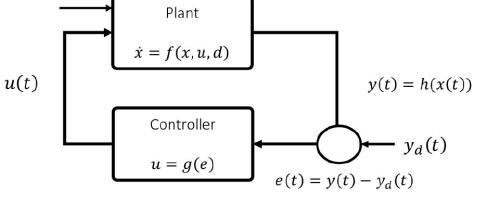
- $\dot{y}(t) = u(t) + d(t)$
- Using proportional (P) controller
- $u(t) = -K_P e(t) = -K_P (y(t) y_d(t))$
- $\dot{y}(t) = -K_P y(t) + K_P y_d(t) + d(t)$
- Consider constant setpoint  $y_0$  and disturbance  $d_{ss}$
- $\dot{y}(t) = -K_P y(t) + K_P y_0 + d_{ss}$
- What is the steady state output?
  - Set  $-K_P y(t) + K_P y_0 + d_{ss} = 0$

• 
$$y(t) = y_{ss} = \frac{d_{ss}}{K_P} + y_0$$



### A simple P-controller example

- $\dot{y}(t) = u(t) + d(t)$
- Consider constant setpoint  $y_0$  and disturbance  $d_{ss}$
- $\cdot \dot{y}(t) = -K_P y(t) + K_P y_0 + d_{ss}$
- Steady state output  $y(t) = y_{ss} = \frac{d_{ss}}{K_P} + y_0$
- Transient behavior
  - $y(t) = y(0)e^{-t/T} + y_{ss}\left(1 e^{-\frac{t}{T}}\right), T = 1/K_P$
- To make steady state error small we can increase  $K_P$  at the expense of longer transients



d(t)

### Summary

- ODE language for control systems
  - Solutions, Lipschitz continuity, equilibria, steady state, transient
- Control design
  - Open loop vs closed loop
  - PID design

